

# Numerical Optimal Control with DAEs

## Lecture 11: High-Index DAEs

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AWESCO PhD course

## Objectives of the lecture

- Why are DAEs not always "easy" to solve ?
- What is a DAE index ? How does it make the DAE "easy" or not ?
- What to do about it ?
- What happens with DAE models for mechanical systems ?
- Some possible additional numerical problems

# Outline

1 "Easy" & "Hard" DAEs

2 Differential Index

3 Index Reduction

4 Constraints drift

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Consider  $\mathbf{F}(\dot{\mathbf{x}}, \mathbf{x}, \mathbf{z}, \mathbf{u}) = 0$  as a **root-finding problem** in  $\dot{\mathbf{x}}$ ,  $\mathbf{z}$ . When can we find  $\dot{\mathbf{x}}$ ,  $\mathbf{z}$ , e.g. using Newton ?

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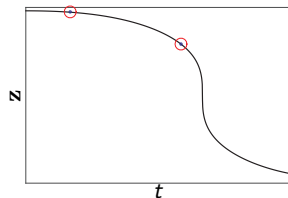
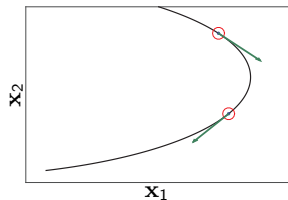
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**Semi-explicit DAE:**

$$\tilde{\mathbf{F}} = \left[ \begin{array}{c} \dot{\mathbf{x}} - \mathbf{F}(\mathbf{z}, \mathbf{x}, \mathbf{u}) \\ \mathbf{G}(\mathbf{z}, \mathbf{x}, \mathbf{u}) \end{array} \right] = 0$$

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$$\begin{bmatrix} \frac{\partial \tilde{\mathbf{F}}}{\partial \dot{\mathbf{x}}} & \frac{\partial \tilde{\mathbf{F}}}{\partial \mathbf{z}} \end{bmatrix} = \begin{bmatrix} \mathbf{I} & \frac{\partial \mathbf{F}}{\partial \mathbf{z}} \\ \mathbf{0} & \frac{\partial \mathbf{G}}{\partial \mathbf{z}} \end{bmatrix}$$

is full-rank if  $\frac{\partial \mathbf{G}}{\partial \mathbf{z}}$  is full rank



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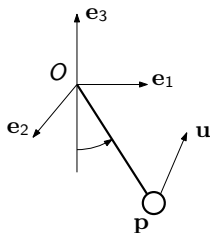
is rank-deficient. This is a "not easy" DAE !! We cannot write  $\dot{\mathbf{x}}_{1,2}$  and  $\mathbf{z}$  as functions of  $\mathbf{x}_{1,2} \dots$



## DAE - 3D pendulum

Model is a **semi-explicit** DAE with  $\mathbf{x} = \begin{bmatrix} \mathbf{p} \\ \mathbf{v} \end{bmatrix}$

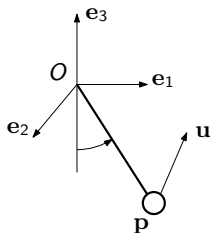
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$$0 = \underbrace{\mathbf{p}^\top \mathbf{p} - L^2}_{\mathbf{G}(\mathbf{x})}$$



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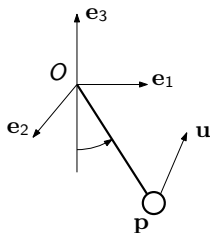
Consider the root-finding problem to be solved in  $\dot{\mathbf{x}}, z$ :

$$\mathbf{r}(\dot{\mathbf{x}}, \mathbf{x}, z, \mathbf{u}) = \begin{bmatrix} \dot{\mathbf{x}} - \mathbf{F}(\mathbf{x}, z, \mathbf{u}) \\ \mathbf{G}(\mathbf{x}) \end{bmatrix} = 0$$

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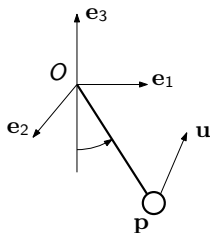
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Then:  $\nabla_{\dot{\mathbf{x}}, z} \mathbf{r}^\top = \begin{bmatrix} I & 0 & 0 \\ 0 & I & \mathbf{p} \\ 0 & 0 & 0 \end{bmatrix}$  is rank-deficient. The Newton step does not exist !!

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Note that  $\frac{\partial \mathbf{G}(\mathbf{x})}{\partial z} = \mathbf{0} !!$

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## DAE - Delta Robot



Lagrange model yields a semi-explicit DAE with:

$$\mathbf{G}(\mathbf{x}) = \begin{bmatrix} \|\mathbf{p} - \mathbf{p}_1\|^2 - L^2 \\ \|\mathbf{p} - \mathbf{p}_2\|^2 - L^2 \\ \|\mathbf{p} - \mathbf{p}_3\|^2 - L^2 \end{bmatrix}$$

where

$$\mathbf{p}_k = R_k^z = \begin{bmatrix} \cos \gamma_k & \sin \gamma_k & 0 \\ -\sin \gamma_k & \cos \gamma_k & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} L \cos \alpha_k \\ 0 \\ L \sin \alpha_k \end{bmatrix}$$

using  $\gamma_{1,2,3} = \{0, \frac{2\pi}{3}, \frac{4\pi}{3}\}$ .

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Such that the DAE:

$$\dot{\mathbf{x}} = \mathbf{F}(\mathbf{x}, \mathbf{z}, \mathbf{u})$$

$$0 = \mathbf{G}(\mathbf{x})$$

... cannot be solved for  $\mathbf{z}$ , because  $\frac{\partial \mathbf{G}(\mathbf{x})}{\partial \mathbf{z}} = 0$  !!

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## DAE from Lagrange Mechanics

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The difficulty comes from having *holonomic* (aka purely position-dependent) constraints:

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$$\frac{d}{dt} \frac{\partial \mathcal{L}}{\partial \dot{\mathbf{q}}} - \frac{\partial \mathcal{L}}{\partial \mathbf{q}} = 0$$

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**What is going on !?!**

# Outline

1 "Easy" & "Hard" DAEs

2 Differential Index

3 Index Reduction

4 Constraints drift

# DAE - Differential Index

### Definition:

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**F is an  
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**An index-1 DAE has**

$$\begin{bmatrix} \frac{\partial \mathbf{F}}{\partial \dot{\mathbf{x}}} & \frac{\partial \mathbf{F}}{\partial \mathbf{z}} \end{bmatrix}$$

**full rank and is therefore "easy" to solve !!**

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For a **semi-explicit DAE** the differential index is the minimum  $i$  such that:

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Example is an **index-2** DAE

## Differential Index - 3D pendulum

**Example:** 3D pendulum

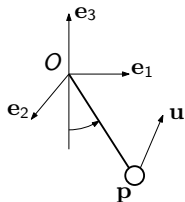
$$m\ddot{\mathbf{p}} = \mathbf{u} - mg\mathbf{e}_3 - z\mathbf{p}$$
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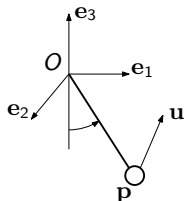
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Substitute  $\ddot{\mathbf{p}}$  from  $m\ddot{\mathbf{p}} = \mathbf{u} - mg\mathbf{e}_3 - z\mathbf{p}$  yields:

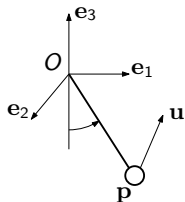
$$\mathbf{p}^\top \left( \frac{1}{m} \mathbf{u} - g\mathbf{e}_3 - \frac{1}{m} z\mathbf{p} \right) + \dot{\mathbf{p}}^\top \dot{\mathbf{p}} = 0$$

For a **semi-explicit DAE**  
the differential index is  
the minimum  $i$  such that:

$$\dot{\mathbf{x}} = \mathbf{F}(\mathbf{x}, \mathbf{z}, \mathbf{u})$$

$$0 = \frac{d^i}{dt^i} \mathbf{G}(\mathbf{x}, \mathbf{z}, \mathbf{u})$$

is an ODE



## Differential Index - 3D pendulum

**Example:** 3D pendulum

$$m\ddot{\mathbf{p}} = \mathbf{u} - mg\mathbf{e}_3 - z\mathbf{p}$$
$$0 = \underbrace{\frac{1}{2}(\mathbf{p}^\top \mathbf{p} - L^2)}_{\mathbf{G}(\mathbf{x})}$$

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i.e.

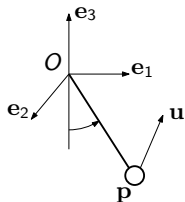
$$z = \frac{1}{\mathbf{p}^\top \mathbf{p}} \left( \mathbf{p}^\top \mathbf{u} - mg\mathbf{p}^\top \mathbf{e}_3 + m\dot{\mathbf{p}}^\top \dot{\mathbf{p}} \right)$$

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A **third** time differentiation yields an ODE for  $z$ :

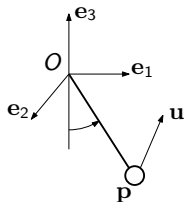
$$\dot{z} = \frac{d}{dt} \left[ \frac{1}{\mathbf{p}^\top \mathbf{p}} \left( \mathbf{p}^\top \mathbf{u} - mg\mathbf{p}^\top \mathbf{e}_3 + m\dot{\mathbf{p}}^\top \dot{\mathbf{p}} \right) \right]$$

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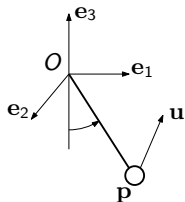
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**The 3D pendulum in Lagrange is an index-3 DAE !!**

## Differential Index - 3D pendulum

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Assemble:

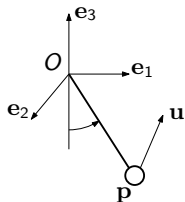
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$$m\ddot{\mathbf{p}} + z\mathbf{p} = \mathbf{u} - mge_3$$
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in matrix form yields:

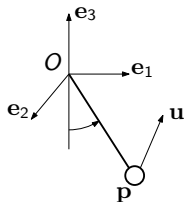
$$\begin{bmatrix} ml & \mathbf{p} \\ \mathbf{p}^\top & 0 \end{bmatrix} \begin{bmatrix} \ddot{\mathbf{p}} \\ z \end{bmatrix} = \begin{bmatrix} \mathbf{u} - mge_3 \\ -\dot{\mathbf{p}}^\top \dot{\mathbf{p}} \end{bmatrix}$$

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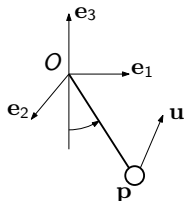
**This is an index-1 (i.e. "easy") DAE !!**

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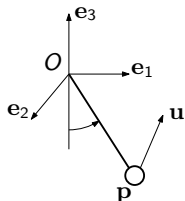
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**We have converted the  
index-3 DAE into an  
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## Differential Index - 3D pendulum

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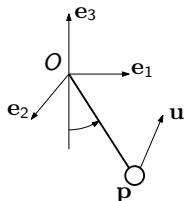
Transforming a high-index DAE into an equivalent lower-index one is labelled **index reduction**

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**We have converted the index-3 DAE into an index-1 DAE !!**

# Outline

1 "Easy" & "Hard" DAEs

2 Differential Index

3 Index Reduction

4 Constraints drift

## DAEs from Lagrange Mechanics

### Index-3 DAE from Lagrange:

$$\frac{d}{d} \frac{\partial \mathcal{L}}{\partial \dot{\mathbf{q}}} - \frac{\partial \mathcal{L}}{\partial \mathbf{q}} = \mathbf{F}_g$$
$$\mathbf{c}(\mathbf{q}) = 0$$

with  $\mathcal{L}(\mathbf{q}, \dot{\mathbf{q}}, \mathbf{z}) = T(\mathbf{q}, \dot{\mathbf{q}}) - V(\mathbf{q}) - \mathbf{z}^T \mathbf{c}(\mathbf{q})$

## DAEs from Lagrange Mechanics

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For most mechanical applications:

$$T(\mathbf{q}, \dot{\mathbf{q}}) = \frac{1}{2} \dot{\mathbf{q}}^\top M(\mathbf{q}) \dot{\mathbf{q}}$$

such that:

$$\frac{d}{d} \frac{\partial \mathcal{L}}{\partial \dot{\mathbf{q}}}^\top = M(\mathbf{q}) \ddot{\mathbf{q}} + \dot{M}(\mathbf{q}, \dot{\mathbf{q}}) \dot{\mathbf{q}}$$

## DAEs from Lagrange Mechanics

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Then the differential part of the DAE model reads as:

$$M(\mathbf{q})\ddot{\mathbf{q}} + \dot{M}(\mathbf{q}, \dot{\mathbf{q}})\dot{\mathbf{q}} - \nabla_{\mathbf{q}}(T(\mathbf{q}, \dot{\mathbf{q}}) - V(\mathbf{q})) + \nabla \mathbf{c}(\mathbf{q}) \mathbf{z} = \mathbf{F}_g$$

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such that:

$$\frac{d}{d} \frac{\partial \mathcal{L}}{\partial \dot{\mathbf{q}}}^T = M(\mathbf{q})\ddot{\mathbf{q}} + \dot{M}(\mathbf{q}, \dot{\mathbf{q}})\dot{\mathbf{q}}$$



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The 1<sup>st</sup> and 2<sup>nd</sup>-order time derivatives of  $\mathbf{c}(\mathbf{q})$  read as:

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### Index-1 DAE model:

$$\begin{bmatrix} M(\mathbf{q}) & \nabla_{\mathbf{q}} \mathbf{c}(\mathbf{q}) \\ \nabla_{\mathbf{q}} \mathbf{c}(\mathbf{q})^\top & 0 \end{bmatrix} \begin{bmatrix} \ddot{\mathbf{q}} \\ \mathbf{z} \end{bmatrix} = \begin{bmatrix} \mathbf{F}_g - \dot{M}(\mathbf{q}, \dot{\mathbf{q}})\dot{\mathbf{q}} + \nabla_{\mathbf{q}}(T(\mathbf{q}, \dot{\mathbf{q}}) - V(\mathbf{q})) \\ -\nabla_{\mathbf{q}} \left( \nabla_{\mathbf{q}} \mathbf{c}(\mathbf{q})^\top \dot{\mathbf{q}} \right)^\top \dot{\mathbf{q}} \end{bmatrix}$$

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Models based on Lagrange mechanics typically are **index-3 DAEs**, making them intrinsically difficult to use. The best approach to treat them is usually to proceed with an **index reduction** down to **index 1** for which very classical integration tools work well.

# Index reduction for semi-explicit DAEs - A general view

## High-index semi-explicit DAE

$$\dot{\mathbf{x}} = \mathbf{F}(\mathbf{x}, \mathbf{z}, \mathbf{u})$$

$$0 = \mathbf{G}(\mathbf{x}, \mathbf{z}, \mathbf{u})$$

**Algorithm** (see "*Nonlinear Programming*", L.T. Biegler)

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- 1 Check if the DAE system is index 1 (i.e.  $\frac{\partial \mathbf{G}}{\partial \mathbf{z}}$  full rank).  
If yes, stop.
- 2 Identify a subset of algebraic equations that **can be solved** for a subset of algebraic variables.
- 3 Apply  $\frac{d}{dt}$  on the remaining algebraic equations that contain the differential variables  $\mathbf{x}_j$ .
- 4 Terms  $\dot{\mathbf{x}}_j$  will appear in these differentiated equations.
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## High-index semi-explicit DAE

$$\dot{\mathbf{x}} = \mathbf{F}(\mathbf{x}, \mathbf{z}, \mathbf{u})$$

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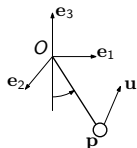
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Writing a general-purpose "Index-reduction algorithm" can be very tricky, as one of the steps is not easily automated



## DAE Consistency - 3D pendulum

Does the index reduction really yield equivalent models ?



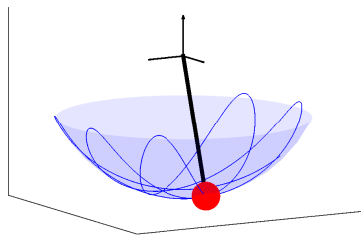
### Index-3 DAE

$$m\ddot{\mathbf{p}} = \mathbf{u} - mge_3 - zp$$

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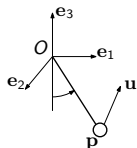
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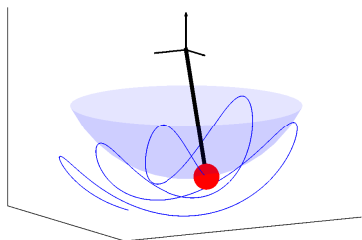
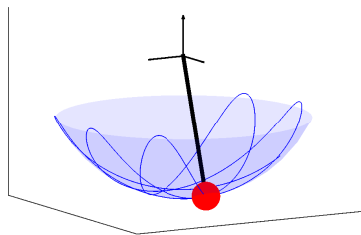


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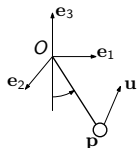
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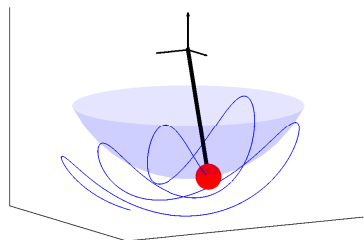
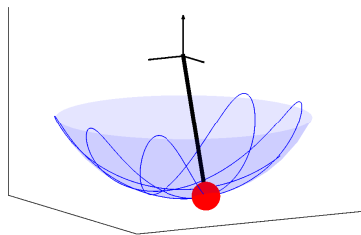


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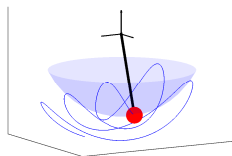
**What is going on ??**

# DAE Consistency - 3D pendulum

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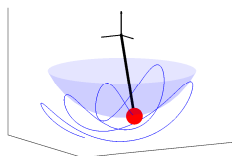
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# DAE Consistency - 3D pendulum

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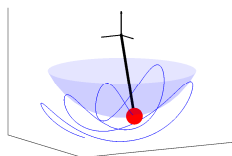
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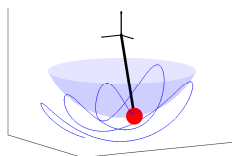
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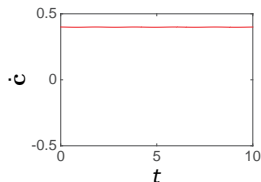
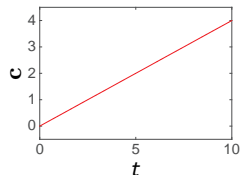
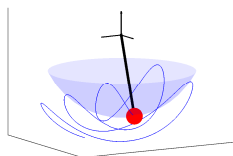
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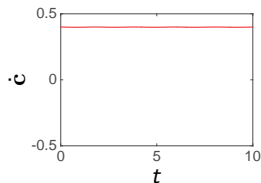
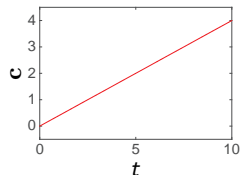
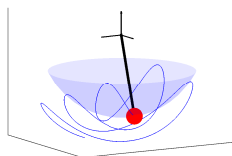
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How can we address that ??

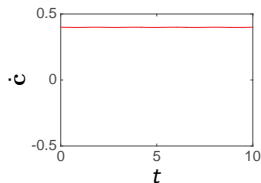
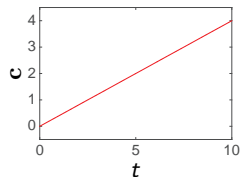
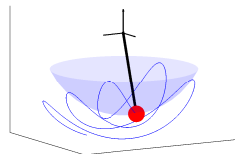


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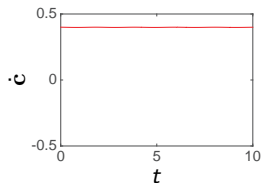
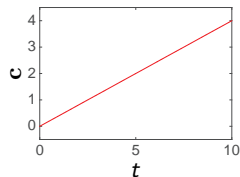
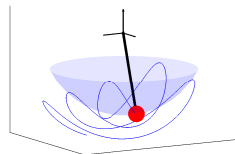
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## DAE Consistency - 3D pendulum

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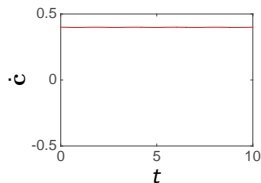
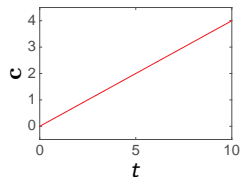
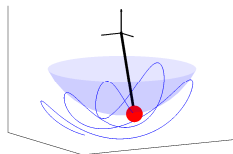
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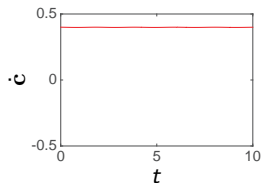
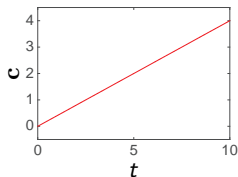
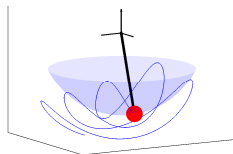
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# DAE Consistency - 3D pendulum

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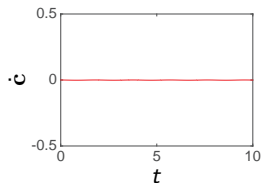
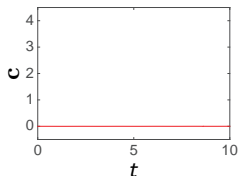
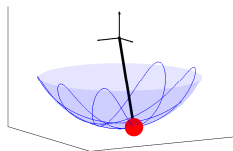
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## Consistency of DAEs from Lagrange Mechanics

### Index-3 DAE from Lagrange:

$$\frac{d}{dt} \frac{\partial \mathcal{L}}{\partial \dot{\mathbf{q}}} - \frac{\partial \mathcal{L}}{\partial \mathbf{q}} = \mathbf{F}_g$$
$$\mathbf{c}(\mathbf{q}) = 0$$

For most mechanical applications:

$$T(\mathbf{q}, \dot{\mathbf{q}}) = \frac{1}{2} \dot{\mathbf{q}}^\top M(\mathbf{q}) \dot{\mathbf{q}}$$

with  $\mathcal{L}(\mathbf{q}, \dot{\mathbf{q}}, \mathbf{z}) = T(\mathbf{q}, \dot{\mathbf{q}}) - V(\mathbf{q}) - \mathbf{z}^\top \mathbf{c}(\mathbf{q})$

Index reduction based on:

$$\frac{d}{dt} \mathbf{c}(\mathbf{q}) = \nabla \mathbf{c}(\mathbf{q})^\top \dot{\mathbf{q}} \quad \text{and} \quad \frac{d^2}{dt^2} \mathbf{c}(\mathbf{q}) = \nabla \mathbf{c}(\mathbf{q})^\top \ddot{\mathbf{q}} + \nabla_{\mathbf{q}} \left( \nabla \mathbf{c}(\mathbf{q})^\top \dot{\mathbf{q}} \right)^\top \dot{\mathbf{q}}$$

### Index-1 DAE model:

$$\begin{bmatrix} M(\mathbf{q}) & \nabla_{\mathbf{q}} \mathbf{c}(\mathbf{q}) \\ \nabla_{\mathbf{q}} \mathbf{c}(\mathbf{q})^\top & 0 \end{bmatrix} \begin{bmatrix} \ddot{\mathbf{q}} \\ \mathbf{z} \end{bmatrix} = \begin{bmatrix} \mathbf{F}_g - \dot{M}(\mathbf{q}, \dot{\mathbf{q}}) \dot{\mathbf{q}} + \nabla_{\mathbf{q}} (T(\mathbf{q}, \dot{\mathbf{q}}) - V(\mathbf{q})) \\ -\nabla_{\mathbf{q}} \left( \nabla \mathbf{c}(\mathbf{q})^\top \dot{\mathbf{q}} \right)^\top \dot{\mathbf{q}} \end{bmatrix}$$

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### Index-1 DAE model:

$$\begin{bmatrix} M(\mathbf{q}) & \nabla_{\mathbf{q}} \mathbf{c}(\mathbf{q}) \\ \nabla_{\mathbf{q}} \mathbf{c}(\mathbf{q})^\top & 0 \end{bmatrix} \begin{bmatrix} \ddot{\mathbf{q}} \\ \mathbf{z} \end{bmatrix} = \begin{bmatrix} \mathbf{F}_g - \dot{M}(\mathbf{q}, \dot{\mathbf{q}}) \dot{\mathbf{q}} + \nabla_{\mathbf{q}} (T(\mathbf{q}, \dot{\mathbf{q}}) - V(\mathbf{q})) \\ -\nabla_{\mathbf{q}} \left( \nabla \mathbf{c}(\mathbf{q})^\top \dot{\mathbf{q}} \right)^\top \dot{\mathbf{q}} \end{bmatrix}$$

with the **consistency conditions**:

$$\mathbf{c}(\mathbf{q}) = 0 \quad \text{and} \quad \frac{d}{dt} \mathbf{c}(\mathbf{q}) = \nabla \mathbf{c}(\mathbf{q})^\top \dot{\mathbf{q}}$$



# Outline

- 1 "Easy" & "Hard" DAEs
- 2 Differential Index
- 3 Index Reduction
- 4 Constraints drift**

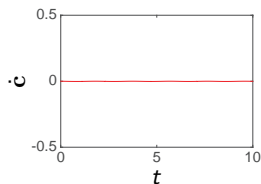
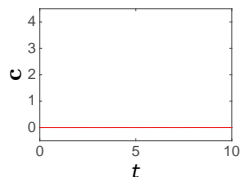
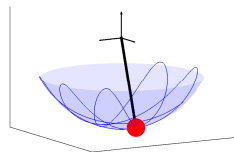
## Constraints drift - 3D pendulum

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... is built to impose  $\ddot{\mathbf{c}} = 0$  at all time.

Then if  $\mathbf{c} = 0$  and  $\dot{\mathbf{c}} = 0$  are satisfied at **any** time on the trajectory, then they are **satisfied at all time**.



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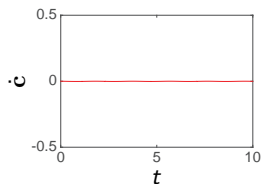
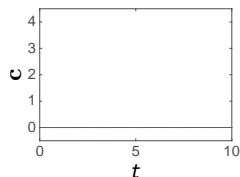
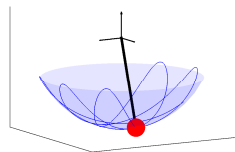
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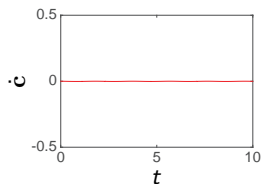
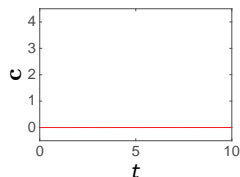
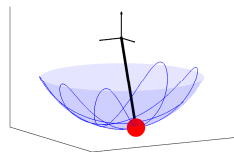
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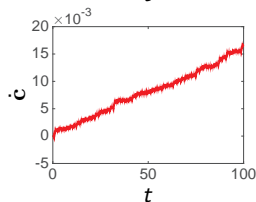
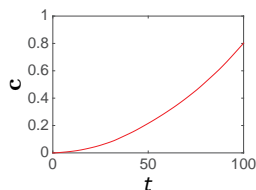
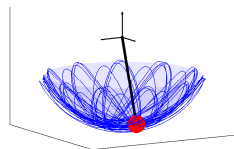
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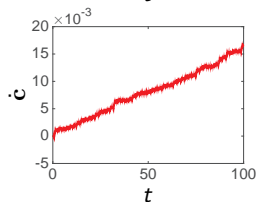
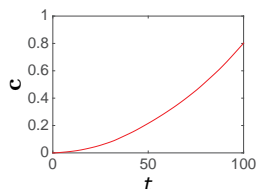
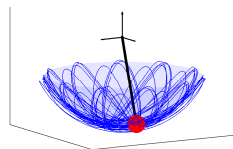
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With consistent initial conditions,  $\mathbf{c} = 0$  and  $\dot{\mathbf{c}} = 0$  **would be** satisfied at all time if we had no numerical error in the integration !!



## Baumgartne stabilization of the constraints drift

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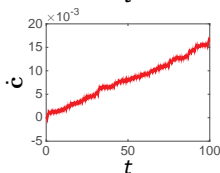
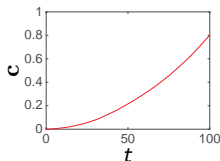
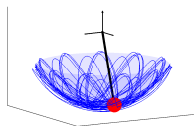
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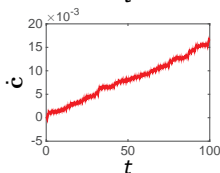
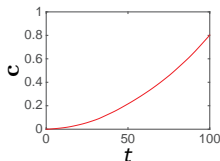
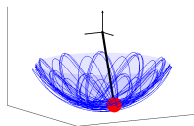
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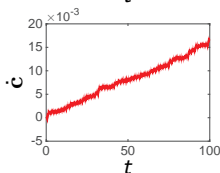
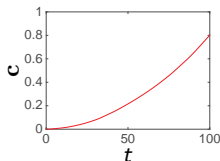
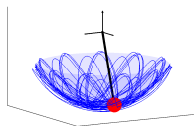
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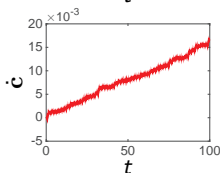
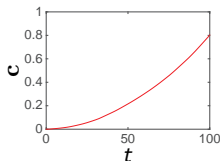
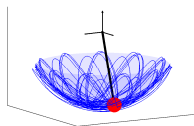
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E.g. for the 3D pendulum:

$$\mathbf{p}^\top \ddot{\mathbf{p}} + \dot{\mathbf{p}}^\top \dot{\mathbf{p}} + \gamma_1 \mathbf{p}^\top \dot{\mathbf{p}} + \frac{\gamma_2}{2} (\mathbf{p}^\top \mathbf{p} - L^2) = 0$$



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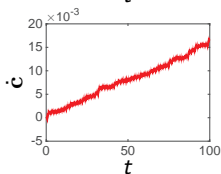
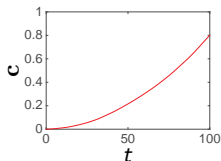
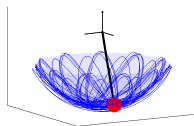
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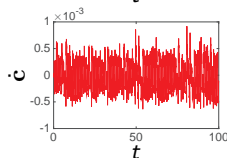
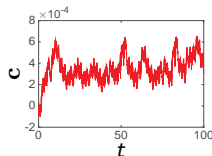
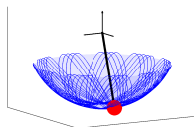
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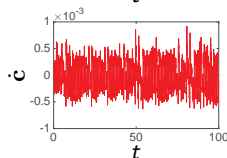
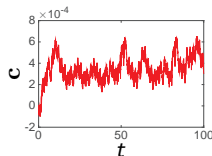
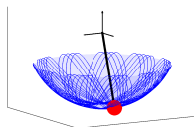
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**The Baumgartne stabilization must be used carefully !**

- Fast poles introduce stiffness in the dynamics
- The interaction between the stabilization and the integrator error is non-trivial...



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Consistency & drift are not DAE-specific.

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This is often not an issue in Optimal Control (reasonably short simulation horizons), but long simulations may require some care.

## A related problem - Invariants in ODEs

Consistency & drift are not DAE-specific.

Simple ODEs

$$\dot{\mathbf{x}} = \mathbf{F}(\mathbf{x}, \mathbf{u})$$

model a physical reality. Some ODEs are representative only when some consistency conditions:

$$\mathbf{c}(\mathbf{x}) = 0$$

are satisfied.

ODEs with consistency conditions occur when one defines a state-space that holds more dimensions than the physical reality it represents !!

Why more states than needed ?

- Simpler, less nonlinear models (this is lifting !!)
- Singularity-free rotations (more on that soon)

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Not covered in this course but good to know: Symplectic integrators (for handling drift)